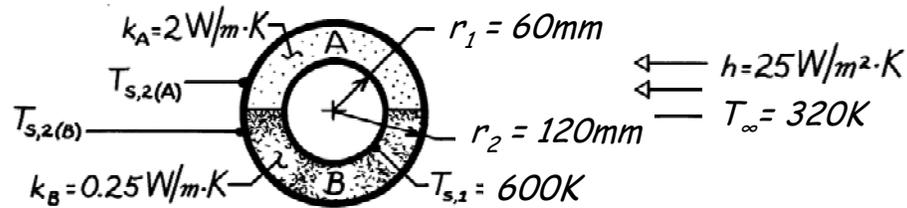


PROBLEM 3.61

KNOWN: Inner surface temperature of insulation blanket comprised of two semi-cylindrical shells of different materials. Ambient air conditions.

FIND: (a) Equivalent thermal circuit, (b) Total heat loss and material outer surface temperatures.

SCHEMATIC:



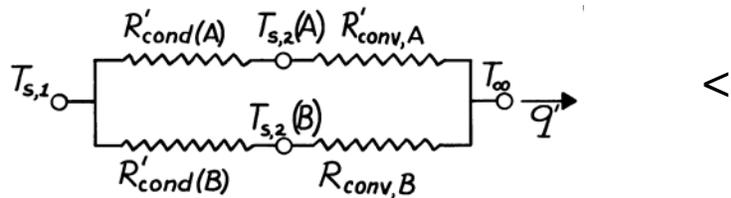
ASSUMPTIONS: (1) Steady-state conditions, (2) One-dimensional, radial conduction, (3) Infinite contact resistance between materials, (4) Constant properties.

ANALYSIS: (a) The thermal circuit is,

$$R'_{\text{conv},A} = R'_{\text{conv},B} = 1/\pi r_2 h$$

$$R'_{\text{cond}(A)} = \frac{\ln(r_2/r_1)}{\pi k_A}$$

$$R'_{\text{cond}(B)} = \frac{\ln(r_2/r_1)}{\pi k_B}$$



The conduction resistances follow from Section 3.3.1 and Eq. 3.33. Each resistance is larger by a factor of 2 than the result of Eq. 3.28 due to the reduced area.

(b) Evaluating the thermal resistances and the heat rate ($q' = q'_A + q'_B$),

$$R'_{\text{conv}} = \left(\pi \times 0.12 \text{ m} \times 25 \text{ W/m}^2 \cdot \text{K} \right)^{-1} = 0.106 \text{ m} \cdot \text{K/W}$$

$$R'_{\text{cond}(A)} = \frac{\ln(0.12 \text{ m}/0.06 \text{ m})}{\pi \times 2 \text{ W/m} \cdot \text{K}} = 0.1103 \text{ m} \cdot \text{K/W} \quad R'_{\text{cond}(B)} = 8 R'_{\text{cond}(A)} = 0.8825 \text{ m} \cdot \text{K/W}$$

$$q' = \frac{T_{s,1} - T_{\infty}}{R'_{\text{cond}(A)} + R'_{\text{conv}}} + \frac{T_{s,1} - T_{\infty}}{R'_{\text{cond}(B)} + R'_{\text{conv}}}$$

$$q' = \frac{(600 - 320) \text{ K}}{(0.1103 + 0.106) \text{ m} \cdot \text{K/W}} + \frac{(600 - 320) \text{ K}}{(0.8825 + 0.106) \text{ m} \cdot \text{K/W}} = (1294 + 283) \text{ W/m} = 1577 \text{ W/m.}$$

Hence, the temperatures are

$$T_{s,2(A)} = T_{s,1} - q'_A R'_{\text{cond}(A)} = 600 \text{ K} - 1294 \frac{\text{W}}{\text{m}} \times 0.1103 \frac{\text{m} \cdot \text{K}}{\text{W}} = 457 \text{ K}$$

$$T_{s,2(B)} = T_{s,1} - q'_B R'_{\text{cond}(B)} = 600 \text{ K} - 283 \frac{\text{W}}{\text{m}} \times 0.8825 \frac{\text{m} \cdot \text{K}}{\text{W}} = 350 \text{ K.}$$

COMMENTS: The total heat loss can also be computed from $q' = (T_{s,1} - T_{\infty})/R_{\text{equiv}}$,

$$\text{where } R_{\text{equiv}} = \left[\left(R'_{\text{cond}(A)} + R'_{\text{conv},A} \right)^{-1} + \left(R'_{\text{cond}(B)} + R'_{\text{conv},B} \right)^{-1} \right]^{-1} = 0.1775 \text{ m} \cdot \text{K/W.}$$

Hence $q' = (600 - 320) \text{ K}/0.1775 \text{ m} \cdot \text{K/W} = 1577 \text{ W/m.}$