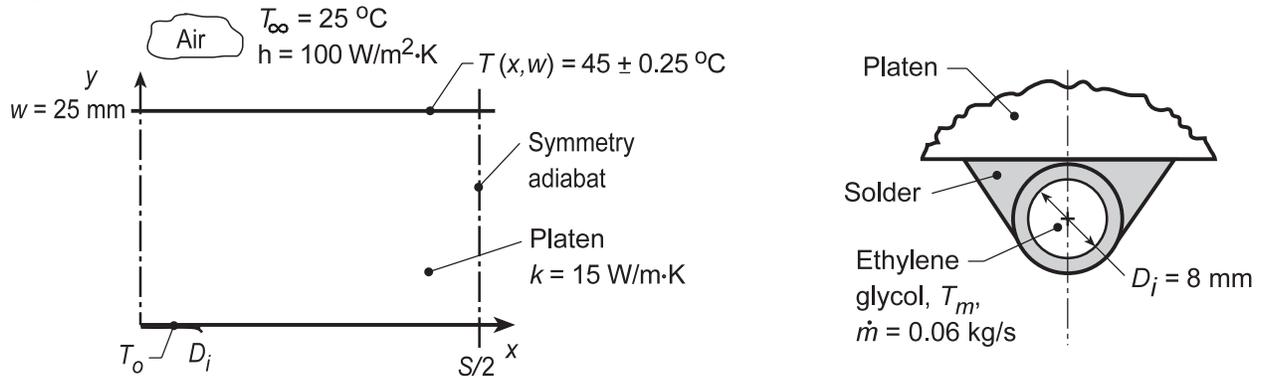


## PROBLEM 8.76

**KNOWN:** Platen heated by hot ethylene glycol flowing through tubing arrangement with spacing  $S$  soldered to lower surface. Top surface exposed to convection process.

**FIND:** Tube spacing  $S$  and heating fluid temperature  $T_m$  which will maintain the top surface at  $45 \pm 0.25^\circ\text{C}$ .

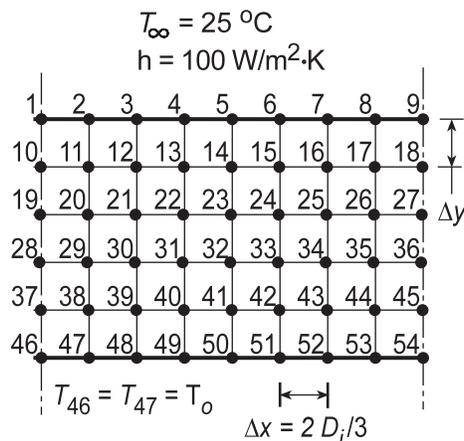
**SCHEMATIC:**



**ASSUMPTIONS:** (1) Steady-state conditions; (2) Lower surface is insulated, all heat transfer from hot fluid is into platen; (3) Copper tube is thick-walled such that interface between solder and platen is isothermal; (4) Fully developed flow conditions in tube.

**PROPERTIES:** Table A.4, Ethylene glycol ( $T_m = 60^\circ\text{C}$ ):  $\mu = 0.00522 \text{ N}\cdot\text{s}/\text{m}^2$ ,  $k = 0.2603 \text{ W}/\text{m}\cdot\text{K}$ .

**ANALYSIS:** Begin the analysis by setting up a nodal mesh ( $9 \times 6$ ) to represent the platen experiencing convection on the top surface ( $T_\infty$ ,  $h$ ) while the two side boundaries are symmetry adiabats. On the lower surface, nodes 46 and 47 represent the isothermal platen-solder interface maintained at  $T_o$  by the hot fluid. The remaining nodes (49-54) are insulated on their lower boundary.



The heat rate supplied by the tube to the platen can be expressed as

$$q'_{cv} = 0.5h_o(\pi D_i)(T_m - T_o) \quad (1)$$

From energy balances about nodes 46 and 47, the heat rate into the platen by conduction can be expressed as

$$q'_{cd} = q'_a + q'_b + q'_c \quad (2)$$

$$q'_a = k(\Delta x/2)(T_{46} - T_{37})/\Delta y \quad (3)$$

Continued...

**PROBLEM 8.76 (Cont.)**

$$q'_b = k(\Delta x)(T_{47} - T_{38})/\Delta y \quad (4)$$

$$q'_c = k(\Delta y/2)(T_{47} - T_{48})/\Delta x \quad (5)$$

and we require that

$$q'_{cd} = q'_{cv} \quad (6)$$

The convection coefficient for internal flow can be estimated from a correlation assuming fully developed flow. First, characterize the flow with

$$Re_D = \frac{4\dot{m}}{\pi D_i \mu} = \frac{4 \times 0.06 \text{ kg/s}}{\pi (0.008 \text{ m}) 0.00522 \text{ N}\cdot\text{s/m}^2} = 1829$$

and since it is laminar,

$$Nu_D = \frac{h_o D_i}{k} = 3.66$$

$$h_o = 3.66 \times 0.2603 \text{ W/m}\cdot\text{K} / 0.008 \text{ m} = 119.1 \text{ W/m}\cdot\text{K}$$

where properties are evaluated at  $T_m$ . Using the *IHT Finite-Difference Tool for Two-Dimensional Steady-State Conditions* and the *Properties Tool for Ethylene Glycol*, along with the foregoing rate equations and energy balances, Eqs. (1-6), a model was developed to solve for the temperature distribution in the platen. In the solution, we determined what hot fluid temperature was required to maintain  $T_1 = 45^\circ\text{C}$ . Two trials were run. In the first, the nodal arrangement was as shown above ( $9 \times 6$ ) for which  $S/2 = (9 - 1)\Delta x = 42.67 \text{ mm}$  with  $\Delta x = 2D_i/3 = 5.33 \text{ mm}$  and  $\Delta y = w/5 = 5 \text{ mm}$ . In the second trial, we repositioned the right-hand symmetry adiabat to pass vertically through the nodes 6-51 so that now the nodal mesh is ( $6 \times 6$ ) and  $S/2 = (6 - 1)\Delta x = 26.65 \text{ mm}$  with  $\Delta x$  and  $\Delta y$  remaining the same. The results of the trials are tabulated below.

Trial	Mesh	$T_1$ ( $^\circ\text{C}$ )	$T_6$ ( $^\circ\text{C}$ )	$T_9$ ( $^\circ\text{C}$ )	$T_m$ ( $^\circ\text{C}$ )	$q'_{cv}$ (W/m)
1	$9 \times 6$	45.0	43.5	43.0	105	80.5
2	$6 \times 6$	45.0	44.5	---	85	52.6

From the trial 2 results, the surface temperature uniformity is  $(T_1 - T_6) = 0.5^\circ\text{C}$  which satisfies the  $\pm 0.25^\circ\text{C}$  requirement. So that suitable tube spacing and fluid temperature are

$$S = 53 \text{ mm}$$

$$T_m = 85^\circ\text{C}$$

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**COMMENTS:** (1) Recognize that the grid spacing is quite coarse and good practice demands that we repeat the analysis decreasing the nodal spacing until no further changes are seen in  $T_m$ .

(2) In the first trial, note that  $T_m = 105^\circ\text{C}$  which of course, is not possible.