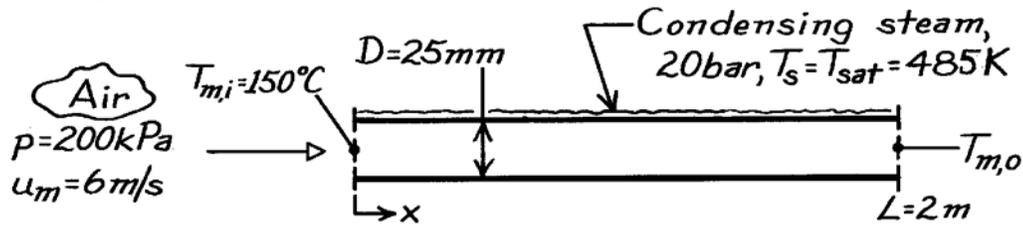


PROBLEM 8.52

KNOWN: Air at prescribed inlet temperature and mean velocity heated by condensing steam on its outer surface.

FIND: (a) Air outlet temperature, pressure drop and heat transfer rate and (b) Effect on parameters of part (a) if pressure were doubled.

SCHEMATIC:



ASSUMPTIONS: (1) Steady-state conditions, (2) Negligible kinetic and potential energy changes, (3) Thermal resistance of tube wall and condensate film are negligible.

PROPERTIES: Table A-4, Air (assume $\bar{T}_m = 450$ K, 1 atm = 101.3 kPa): $\rho = 0.7740$ kg/m³, $c_p = 1021$ J/kg·K, $\mu = 250.7 \times 10^{-7}$ N·s/m², $k = 0.0373$ W/m·K, $Pr = \mu c_p / k = 0.686$. Note that only ρ is pressure dependent; i.e., $\rho \propto p$; Table A-6, Saturated water (20 bar): $T_{sat} = T_s = 485$ K.

ANALYSIS: (a) For constant wall temperature heating, from Eq. 8.41b,

$$\frac{T_s - T_{m,o}}{T_s - T_{m,i}} = \exp\left(-\frac{PL}{\dot{m} c_p} \bar{h}_i\right)$$

where $P = \pi D$. For the air flow, find the mass rate and Reynolds number,

$$\dot{m} = \rho A_c u_m = 0.7740 \text{ kg/m}^3 (200 \text{ kPa}/101.3 \text{ kPa}) \left(\pi (0.025 \text{ m})^2 / 4\right) \times 6 \text{ m/s}$$

$$\dot{m} = 4.501 \times 10^{-3} \text{ kg/s.}$$

$$Re_D = \frac{4\dot{m}}{\mu \pi D} = \frac{4 \times 4.501 \times 10^{-3} \text{ kg/s}}{250.7 \times 10^{-7} \text{ N}\cdot\text{s/m}^2 \times \pi (0.025 \text{ m})} = 9.143 \times 10^3.$$

Using the Dittus-Boelter correlation for fully-developed turbulent flow,

$$Nu_D = 0.023 Re^{4/5} Pr^{0.4} = 0.023 (9.143 \times 10^3)^{4/5} (0.682)^{0.4} = 29.12$$

$$h_i = Nu \cdot k / D = 29.12 \times 0.0373 \text{ W/m}\cdot\text{K} / 0.025 \text{ m} = 43.4 \text{ W/m}^2 \cdot \text{K}.$$

Hence, the outlet temperature is

$$\frac{212 - T_{m,o}}{(212 - 150)^\circ \text{C}} = \exp\left[-\frac{\pi (0.025 \text{ m}) \times 2 \text{ m} \times 43.4 \text{ W/m}^2 \cdot \text{K}}{4.501 \times 10^{-3} \text{ kg/s} \times 1021 \text{ J/kg}\cdot\text{K}}\right]$$

$$T_{m,o} = 198^\circ \text{C.}$$

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Continued ...

PROBLEM 8.52 (Cont.)

The pressure drop follows from Eqs. 8.21 and 8.22,

$$f = (0.790 \ln(\text{Re}_D) - 1.64)^{-2} = \left(0.790 \ln(9.143 \times 10^3) - 1.64\right)^{-2} = 0.0323$$

$$\Delta p = f \frac{\rho u_m^2}{2D} L$$

$$\Delta p = 0.0323 \frac{0.7740 \text{ kg/m}^3 (200/101.3) (6 \text{ m/s})^2 \times 2 \text{ m}}{2 \times 0.025 \text{ m}} = 71.1 \text{ N/m}^2. \quad <$$

The heat transfer rate is

$$q = \dot{m} c_p (T_{m,o} - T_{m,i}) = 4.501 \times 10^{-3} \text{ kg/s} \times 1021 \text{ J/kg} \cdot \text{K} (198 - 150) \text{ K} = 221 \text{ W}. \quad <$$

(b) If the pressure were doubled, we can see from the above relations that $\dot{m} \propto p$ hence

$$\dot{m} = 2\dot{m}_o$$

$$\text{Re}_D = 2\text{Re}_{D,o},$$

since

$$h_i \propto (\text{Re})^{4/5} \rightarrow (h_i / h_{i,o}) = 2^{4/5},$$

$$h_i = 1.74 h_{i,o}.$$

It follows that $T_{m,o} = 195^\circ\text{C}$, so that the effect on temperature is slight. However, the pressure drop increases by the factor 1.68 and the heat rate by the factor 1.88. In summary:

Parameter	p = 200 kPa Part (a)	p = 400 kPa Part (b)	Increase, %
\dot{m} , kg/s $\times 10^3$	4.501	9.002	100
h_i , W/m ² ·K	43.4	75.6	74
$T_{m,o} - T_{m,i}$, °C	48	45	-6
Δp , N/m ²	71.1	119	68
q, W	221	415	88

COMMENTS: (1) Note that $\bar{T}_m = (198 + 150)^\circ\text{C}/2 = 447 \text{ K}$ agrees well with the assumed value (450 K) used to evaluate the thermophysical properties.