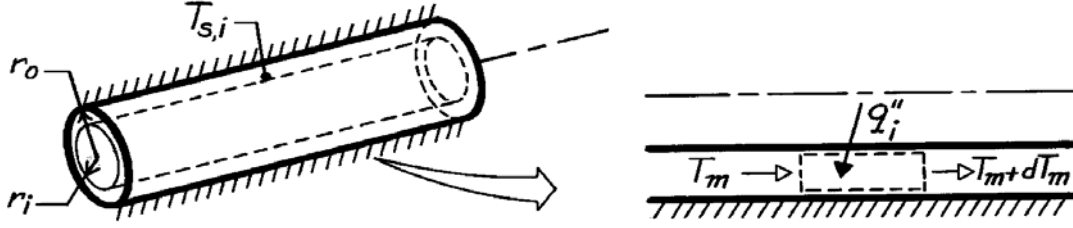


PROBLEM 8.95

KNOWN: Inner and outer tube surface conditions for an annulus.

FIND: (a) Velocity profile, (b) Temperature profile and expression for inner surface Nusselt number.

SCHEMATIC:



ASSUMPTIONS: (1) Steady-state conditions, (2) Laminar, fully developed flow, (3) Uniform heat flux at inner surface, (4) Adiabatic outer surface, (5) Constant properties, (6) Applicability of Eq. 8.34.

ANALYSIS: (a) From Section 8.1.3, the general solution to Eq. 8.12, which also applies to annular flow as represented in Figure 8.11, is

$$u(r) = \frac{1}{\mu} \left(\frac{dp}{dx} \right) \frac{r^2}{4} + C_1 \ln r + C_2.$$

Applying the boundary conditions,

$$u(r_i) = 0 \quad 0 = \frac{1}{\mu} \left(\frac{dp}{dx} \right) \frac{r_i^2}{4} + C_1 \ln r_i + C_2$$

$$u(r_o) = 0 \quad 0 = \frac{1}{\mu} \left(\frac{dp}{dx} \right) \frac{r_o^2}{4} + C_1 \ln r_o + C_2.$$

Hence,

$$C_1 = \frac{\frac{1}{\mu} \left(\frac{dp}{dx} \right) \left(\frac{r_o^2}{4} - \frac{r_i^2}{4} \right)}{\ln r_i / r_o} \quad C_2 = -\frac{1}{\mu} \left(\frac{dp}{dx} \right) \frac{r_o^2}{4} - \frac{1}{\mu} \left(\frac{dp}{dx} \right) \left(\frac{r_o^2}{4} - \frac{r_i^2}{4} \right) \frac{\ln r_o}{\ln (r_i / r_o)}$$

and the velocity distribution is

$$\begin{aligned} u(r) &= \frac{1}{\mu} \left(\frac{dp}{dx} \right) \left(\frac{r^2}{4} - \frac{r_o^2}{4} \right) + \frac{1}{\mu} \left(\frac{dp}{dx} \right) \left(\frac{r_o^2}{4} - \frac{r_i^2}{4} \right) \frac{\ln r}{\ln (r_i / r_o)} \\ &\quad - \frac{1}{\mu} \left(\frac{dp}{dx} \right) \left(\frac{r_o^2}{4} - \frac{r_i^2}{4} \right) \frac{\ln r_o}{\ln (r_i / r_o)} \\ u(r) &= -\frac{r_o^2}{4\mu} \left(\frac{dp}{dx} \right) \left[1 - (r/r_o)^2 + \frac{(r_i/r_o)^2 - 1}{\ln (r_i / r_o)} \ln (r/r_o) \right]. \end{aligned} \quad (1) <$$

(b) For fully developed conditions with uniform surface heat flux,

$$v = 0 \quad \partial T / \partial x = dT_m / dx = \text{const.}$$

Continued ...

PROBLEM 8.95 (Cont.)

Hence, from Eq. 8.48, which also applies for annular flow,

$$\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) = \frac{u}{\alpha} \frac{dT_m}{dx}$$

Substituting the velocity distribution, with

$$C_1 = -\frac{r_o^2}{4\mu} \left(\frac{dp}{dx} \right) \quad C_2 = \frac{(r_i/r_o)^2 - 1}{\ln(r_i/r_o)} \quad (2)$$

it follows that $\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right) = \frac{C_1}{\alpha} \frac{dT_m}{dx} \left[1 - (r/r_o)^2 + C_2 \ln(r/r_o) \right]$.

$$r \frac{\partial T}{\partial r} = \frac{C_1}{\alpha} \frac{dT_m}{dx} \int \left[r - \frac{r^3}{r_o^2} + C_2 r \ln \frac{r}{r_o} \right] dr + C_3$$

$$\frac{\partial T}{\partial r} = \frac{C_1}{\alpha} \frac{dT_m}{dx} \left[\frac{r}{2} - \frac{r^3}{4r_o^2} + C_2 \left(\frac{r}{2} \ln \frac{r}{r_o} - \frac{r}{4} \right) \right] + \frac{C_3}{r}$$

and the temperature distribution is

$$T(r) = \frac{C_1}{\alpha} \frac{dT_m}{dx} \left[\frac{r^2}{4} - \frac{r^4}{16r_o^2} + C_2 \left(\frac{r^2}{4} \ln \frac{r}{r_o} - \frac{r^2}{4} \right) \right] + C_3 \ln r + C_4. \quad (3) <$$

From the requirement that $q_o'' = 0$, it follows that $\partial T / \partial r|_{r_o} = 0$. Hence,

$$\frac{C_1}{\alpha} \frac{dT_m}{dx} \left[\frac{r_o}{2} - \frac{r_o}{4} + C_2 \left(-\frac{r_o}{4} \right) \right] + \frac{C_3}{r_o} = 0$$

$$C_3 = \frac{C_1}{\alpha} \frac{dT_m}{dx} \frac{r_o^2}{4} (C_2 - 1). \quad (4) <$$

From the condition that $T(r_i) = T_{s,i}$, it follows that

$$C_4 = T_{s,i} - \frac{C_1}{\alpha} \frac{dT_m}{dx} \left[\frac{r_i^2}{4} - \frac{r_i^4}{16r_o^2} + C_2 \left(\frac{r_i^2}{4} \ln \frac{r_i}{r_o} - \frac{r_i^2}{4} \right) \right] + C_3 \ln r_i. \quad (5) <$$

From Eqs. 8.67 and 8.69, the inner surface Nusselt number is

$$Nu_i = \frac{h_i D_h}{k} = \frac{q_i'' D_h}{k(T_{s,i} - T_m)}$$

where $D_h = 2(r_o - r_i)$. To obtain a workable form of Nu_i , the mean temperature T_m must be evaluated.

This may be done by substituting Eqs. (1) and (3) into Eq. 8.26 and evaluating u_m by substituting Eq. (1) into Eq. 8.8. Since the integrations are long and tedious, they are not provided.

COMMENTS: From an energy balance performed for a differential control volume in the annular region, $dT_m/dx = 2r_i q_i'' / \rho c_p u_m (r_o^2 - r_i^2)$.