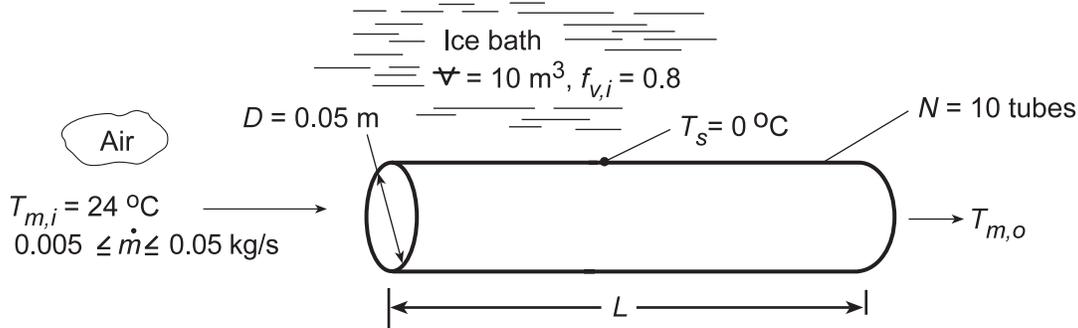


PROBLEM 8.33

KNOWN: Diameter and surface temperature of ten tubes in an ice bath. Inlet temperature and flowrate per tube. Volume (∇) of container and initial volume fraction, $f_{v,i}$, of ice.

FIND: (a) Tube length required to achieve a prescribed air outlet temperature $T_{m,o}$ and time to completely melt the ice, (b) Effect of mass flowrate on $T_{m,o}$ and suitable design and operating conditions.

SCHEMATIC:



ASSUMPTIONS: (1) Steady-state, (2) Ideal gas with negligible viscous dissipation and pressure variation, (3) Constant properties, (4) Fully developed flow throughout each tube, (5) Negligible tube wall thermal resistance.

PROPERTIES: Table A.4, air (assume $\bar{T}_m = 292$ K): $c_p = 1007$ J/kg·K, $\mu = 180.6 \times 10^{-7}$ N·s/m², $k = 0.0257$ W/m·K, $Pr = 0.709$; Ice: $\rho = 920$ kg/m³, $h_{sf} = 3.34 \times 10^5$ J/kg.

ANALYSIS: (a) With $Re_D = 4 \dot{m} / \pi D \mu = 4(0.01 \text{ kg/s}) / \pi(0.05 \text{ m})180.6 \times 10^{-7} \text{ N}\cdot\text{s/m}^2 = 14,100$ for $\dot{m} = 0.01$ kg/s, the flow is turbulent, and from Eq. 8.60,

$$\overline{Nu}_D = Nu_D = 0.023 Re_D^{0.8} Pr^{0.3} = 0.023(14,100)^{0.8} (0.709)^{0.3} = 43.3$$

$$\bar{h} = \overline{Nu}_D (k/D) = 43.3(0.0257 \text{ W/m}\cdot\text{K}/0.05 \text{ m}) = 22.2 \text{ W/m}^2\cdot\text{K}$$

With $T_{m,o} = 14^\circ\text{C}$, the tube length may be obtained from Eq. 8.41b,

$$\frac{T_s - T_{m,o}}{T_s - T_{m,i}} = \frac{-14}{-24} = \exp\left(-\frac{\pi DL \bar{h}}{\dot{m} c_p}\right) = \exp\left[-\frac{\pi(0.05 \text{ m})(22.2 \text{ W/m}^2\cdot\text{K})L}{0.01 \text{ kg/s}(1007 \text{ J/kg}\cdot\text{K})}\right]$$

$$L = 1.56 \text{ m} \quad \leftarrow$$

The time required to completely melt the ice may be obtained from an energy balance of the form,

$$(-q)t = f_{v,i} \nabla (\rho h_{sf})$$

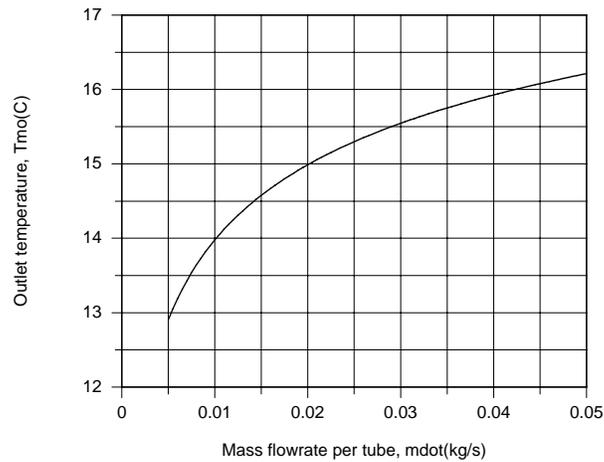
where $q = N \dot{m} c_p (T_{m,i} - T_{m,o}) = 10(0.01 \text{ kg/s})1007 \text{ J/kg}\cdot\text{K}(10 \text{ K}) = 1007 \text{ W}$. Hence,

$$t = \frac{0.8(10 \text{ m}^3)(920 \text{ kg/m}^3)3.34 \times 10^5 \text{ J/kg}}{1007 \text{ W}} = 2.44 \times 10^6 \text{ s} = 28.3 \text{ days} \quad \leftarrow$$

(b) Using the appropriate IHT Correlations and Properties Tool Pads, the following results were obtained.

Continued...

PROBLEM 8.33 (Cont.)



Although heat extraction from the air passing through each tube increases with increasing flowrate, the increase is not in proportion to the change in \dot{m} and the temperature difference ($T_{m,i} - T_{m,o}$) decreases. If 0.05 kg/s of air is routed through a single tube, the outlet temperature of $T_{m,o} = 16.2^\circ\text{C}$ slightly exceeds the desired value of 16°C . The prescribed value could be achieved by slightly increasing the tube length. However, in the interest of reducing pressure drop requirements, it would be better to operate at a lower flowrate per tube. If, for example, air is routed through four of the tubes at 0.01 kg/s per tube and the discharge is mixed with 0.01 kg/s of the available air at 24°C , the desired result would be achieved.

COMMENTS: Since the flow is turbulent and $L/D = 31$, the assumption of fully developed flow throughout a tube is marginal and the foregoing analysis overestimates the discharge temperature.