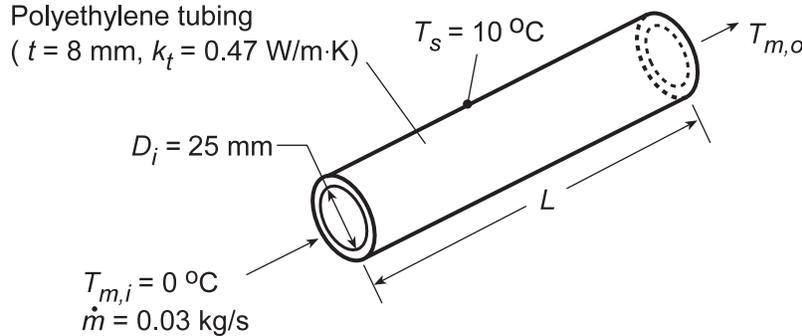


## PROBLEM 8.77

**KNOWN:** Features of tubing used in a ground source heat pump. Temperature of surrounding soil. Fluid inlet temperature and flowrate.

**FIND:** (a) Effect of tube length on outlet temperature, (b) Recommended tube length and the effect of variations in the flowrate.

**SCHEMATIC:**



**ASSUMPTIONS:** (1) Steady-state conditions, (2) Constant properties, (3) Negligible conduction resistance in soil, (4) Incompressible liquid with negligible viscous dissipation, (5) Fluid properties correspond to those of water.

**PROPERTIES:** Table A.6 (assume  $\bar{T}_m = 277 \text{ K}$ ):  $c_p = 4206 \text{ J/kg}\cdot\text{K}$ ,  $\mu = 1560 \times 10^{-6} \text{ N}\cdot\text{s/m}^2$ ,  $k = 0.577 \text{ W/m}\cdot\text{K}$ ,  $\text{Pr} = 11.44$ .

**ANALYSIS:** (a) For the prescribed conditions,  $\text{Re}_D = 4\dot{m}/\pi D_i \mu = 4(0.03 \text{ kg/s})/\pi(0.025 \text{ m})1560 \times 10^{-6} \text{ N}\cdot\text{s/m}^2 = 980$  and the flow is laminar. With  $\text{Pr} > 5$ , Eq. 8.57 may be used to determine the average convection coefficient, with Eq. 8.56 defining the Graetz number:

$$\overline{\text{Nu}}_D = 3.66 + \frac{0.0668(D/L)\text{Re}_D \text{Pr}}{1 + 0.04[(D/L)\text{Re}_D \text{Pr}]^{2/3}}$$

With  $T_s$  used in lieu of  $T_\infty$ , Eq. 8.45b may be used to determine  $T_{m,o}$ ,

$$\frac{T_s - T_{m,o}}{T_s - T_{m,i}} = \exp\left(-\frac{L}{\dot{m}c_p R'_{\text{tot}}}\right)$$

where  $R'_{\text{tot}}$  accounts for the convection and tube wall conduction resistances,

$$R'_{\text{tot}} = R'_{\text{cnv}} + R'_{\text{cnd}} = \left(1/\pi D_i \bar{h}\right) + \ln(D_o/D_i)/2\pi k_t$$

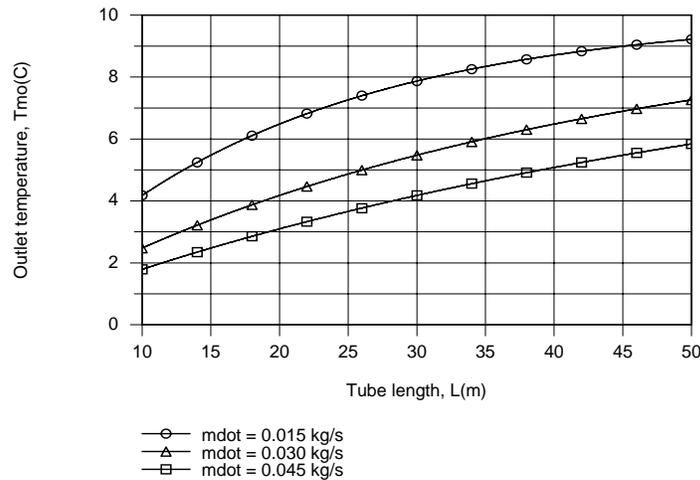
and

$$D_o = D_i + 2t = 41 \text{ mm}.$$

Using the *Correlations and Properties* Toolpads of IHT, the following results were obtained for the effect of the tube length  $L$  on  $T_{m,o}$ .

Continued...

### PROBLEM 8.77 (Cont.)



The longer the tube the larger the rate of heat extraction from the soil, and for  $\dot{m} = 0.030$  kg/s, the temperature rise of  $\Delta T = (T_{m,o} - T_{m,i}) \approx 7^\circ\text{C}$  is well below the maximum possible value of  $\Delta T_{\max} = 10^\circ\text{C}$ .

(b) The length should be *at least* 50 m long. If the flowrate were reduced by 50% ( $\dot{m} = 0.015$  kg/s), the corresponding temperature rise would be close to  $\Delta T_{\max}$  and  $L = 50$  m would be close to optimal. However, for the nominal flowrate and a 50% increase from the nominal, the length should exceed 50 m to recover more heat and provide a heat pump inlet temperature which is closer to the maximum possible value.

**COMMENTS:** In practice, the tube surface temperature would be less than  $10^\circ\text{C}$  (if the temperature of the soil well removed from the tube were at  $10^\circ\text{C}$ ), thereby reducing the heat extraction rate and  $T_{m,o}$ .