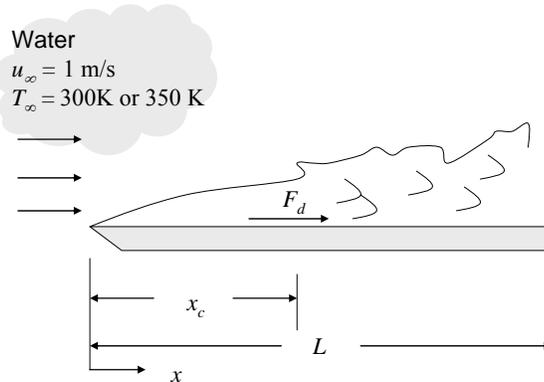


## PROBLEM 6.47

**KNOWN:** Velocity of water flowing over a flat plate. Length and width of plate. Variation of local convection coefficient with  $x$  for  $T = 300\text{ K}$  and  $T = 350\text{ K}$ . Locations of turbulence transition.

**FIND:** Drag force for both water temperatures.

**SCHEMATIC:**



**ASSUMPTIONS:** (1) Steady-state conditions, (2) Transition occurs at a critical Reynolds number of  $5 \times 10^5$ , (3) Incompressible flow.

**PROPERTIES:** Table A.6, Water ( $T = 300\text{ K}$ ):  $\mu = 855 \times 10^{-6}\text{ N}\cdot\text{s}/\text{m}^2$ ,  $k = 0.613\text{ W}/\text{m}\cdot\text{K}$ . Water ( $T = 350\text{ K}$ ):  $\mu = 365 \times 10^{-6}\text{ N}\cdot\text{s}/\text{m}^2$ ,  $k = 0.668\text{ W}/\text{m}\cdot\text{K}$ .

**ANALYSIS:** According to the Reynolds analogy, Eq. 6.35

$$C_f \frac{Re_L}{2} = Nu$$

This relationship holds for the local values of  $C_f$  and  $Nu$ . The local shear stress can be expressed as

$$\tau_s = C_f \frac{\rho u_\infty^2}{2} = \frac{Nu}{Re_L} \rho u_\infty^2 = \frac{hx}{k} \frac{\rho u_\infty \nu}{L} = \frac{\mu u_\infty}{kL} hx = Bhx$$

where  $B = \mu u_\infty / kL$ . Therefore,  $\tau_{s,\text{lam}} = BC_{\text{lam}} x^{0.5}$  and  $\tau_{s,\text{turb}} = BC_{\text{turb}} x^{0.8}$ . Now the drag force can be found:

$$F_d = \int_0^L \tau_s dx \cdot W = \left[ \int_0^{x_c} BC_{\text{lam}} x^{0.5} dx + \int_{x_c}^L BC_{\text{turb}} x^{0.8} dx \right] W = BW \left[ C_{\text{lam}} \frac{x_c^{1.5}}{1.5} + C_{\text{turb}} \left( \frac{L^{1.8}}{1.8} - \frac{x_c^{1.8}}{1.8} \right) \right]$$

At  $T = 300\text{ K}$ ,

$$F_d = \frac{855 \times 10^{-6}\text{ N}\cdot\text{s}/\text{m}^2 \times 1\text{ m}/\text{s}}{0.613\text{ W}/\text{m}\cdot\text{K} \times 0.6\text{ m}} \times 1\text{ m} \\ \times \left[ 395\text{ W}/\text{m}^{1.5} \cdot \text{K} \times \frac{(0.43\text{ m})^{1.5}}{1.5} + 2330\text{ W}/\text{m}^{1.8} \cdot \text{K} \left( \frac{(0.6\text{ m})^{1.8}}{1.8} - \frac{(0.43\text{ m})^{1.8}}{1.8} \right) \right]$$

$$F_d = 0.714\text{ N}$$

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Continued...

### PROBLEM 6.47 (Cont.)

Similarly, at  $T = 350$  K with  $C_{\text{lam}} = 477 \text{ W/m}^{1.5}\cdot\text{K}$ ,  $C_{\text{turb}} = 3600 \text{ W/m}^{1.8}\cdot\text{K}$  and  $x_c = 0.19$  m,

$$F_d = 0.659 \text{ N}$$

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**COMMENTS:** (1) Even though transition to turbulence occurs earlier for the  $T = 350$  K case, the net effect of the much smaller viscosity is a reduction in the drag force. (2) It would be incorrect to apply Reynolds' analogy, Eq. 6.35, directly to the average values of  $C_f$  and  $Nu$  because of the presence of  $x$  in the definition of the Nusselt number. Applying Eq. 6.35 directly to the average values would result in the incorrect values  $F_d = 1.36$  and  $1.22$  N for the 300 K and 350 K cases, respectively.